

GA-A23494

THERMAL HYDRAULIC ANALYSIS OF FIRE DIVERTOR

by
C.B. BAXI, M.A. ULRICKSON, D.E. DRIEMEYER,
and P. HEITZENROEDER

OCTOBER 2000

DISCLAIMER

This report was prepared as an account of work sponsored by an agency of the United States Government. Neither the United States Government nor any agency thereof, nor any of their employees, makes any warranty, express or implied, or assumes any legal liability or responsibility for the accuracy, completeness, or usefulness of any information, apparatus, product, or process disclosed, or represents that its use would not infringe privately owned rights. Reference herein to any specific commercial product, process, or service by trade name, trademark, manufacturer, or otherwise, does not necessarily constitute or imply its endorsement, recommendation, or favoring by the United States Government or any agency thereof. The views and opinions of authors expressed herein do not necessarily state or reflect those of the United States Government or any agency thereof.

THERMAL HYDRAULIC ANALYSIS OF FIRE DIVERTOR

by
C.B. BAXI, M.A. ULRICKSON,[†] D.E. DRIEMEYER,[‡]
and P. HEITZENROEDER[◇]

This is a preprint of a paper presented at the 14th Topical Meeting on the Technology of Fusion Energy, October 15–19, 2000 in Park City, Utah and to be published in *Fusion Technology*.

[†]Sandia National Laboratories

[‡]Boeing

[◇]Princeton Plasma Physics Laboratory

Work supported by
the U.S. Department of Energy under
Contract Nos. DE-AC03-98ER54411, DE-AC02-76CH03073,
and DE-AC04-94AL85000

GA PROJECT 30007
OCTOBER 2000

THERMAL HYDRAULIC ANALYSIS OF FIRE DIVERTOR

C.B. Baxi,¹ M.A. Ulrickson,² D.E. Driemeyer,³ and P. Heitzenroeder⁴

¹General Atomics, P.O. Box 85608, San Diego, California 92186-5608

²Sandia National Laboratories, P.O. Box 5800, Albuquerque, New Mexico 87185

³Boeing, P.O. Box 516, St. Louis, Missouri 63166-0516

⁴Princeton Plasma Physics Laboratory, P.O. Box 451, Princeton, New Jersey 08543-0451

ABSTRACT

The Fusion Ignition Research Experiment (FIRE) is being designed as a next step in the U.S. magnetic fusion program. The FIRE tokamak has a major radius of 2 m, a minor radius of 0.525 m, and liquid nitrogen cooled copper coils. The aim is to produce a pulse length of 20 s with a plasma current of 6.6 MA and with alpha dominated heating.

The outer divertor and baffle of FIRE are water cooled. The worst thermal condition for the outer divertor and baffle is the baseline D-T operating mode (10 T, 6.6 MA, 20 s) with a plasma exhaust power of 67 MW and a peak heat flux of 20 MW/m². A swirl tape (ST) heat transfer enhancement method is used in the outer divertor cooling channels to increase the heat transfer coefficient and the critical heat flux (CHF). The plasma-facing surface consists of tungsten brush.

The finite element (FE) analysis shows that for an inlet water temperature of 30°C, inlet pressure of 1.5 MPa and a flow velocity of 10 m/s, the incident critical heat flux is greater than 30 MW/m². The peak copper temperature is 490°C, peak tungsten temperature is 1560°C, and the pressure drop is less than 0.5 MPa. All these results fulfill the design requirements.

I. INTRODUCTION

The Fusion Ignition Research Experiment (FIRE) is being designed for high power density and advanced physics operating modes.¹ The FIRE has a double-null divertor configuration. The baffle and outer divertor are actively water cooled. The inner divertor has low heat flux and is cooled by conduction to the copper shell inside the

vessel wall. Figure 1 shows the cross section of the FIRE with location of inner and outer divertor and baffle.

II. DIVERTOR AND BAFFLE GEOMETRY

The divertor and baffle design of the FIRE is based on technologies developed for ITER.² There are 32 modules each of the divertor and baffle (16 upper and 16 lower). A module is divided into 24 copper (Cu-Cr-Zr) plates across the front surface. The copper plates include tungsten-brush armor as a plasma-facing component (PFC) and coolant channels on both divertor and baffle. The tungsten rods are 3 mm in diameter arranged on a triangular pitch of 3.1 mm. The rods of the brush have a conical tip over 1 mm length on the heat sink side. The tungsten rods are joined to the copper with HIP-bonding process.² A 5 mm thickness of the PFC gives adequate lifetime under the expected disruption conditions. The use of tungsten brush reduces the stresses in the PFC. This is

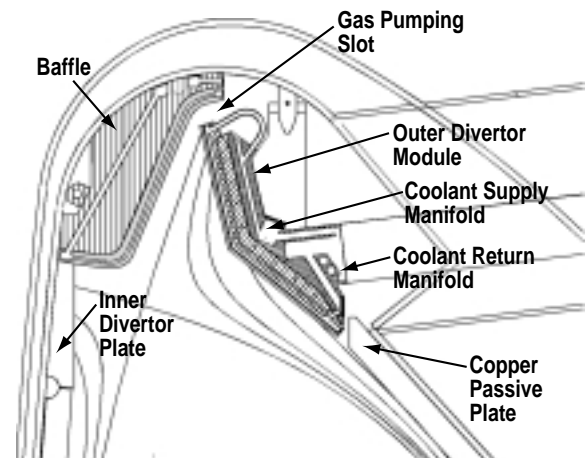


Fig. 1. Cross Section Through FIRE divertor and baffle.

one of the designs for the ITER divertor PFC in the highest heat flux region. This PFC has been successfully fabricated and tested for ITER and found suitable for heat fluxes upto 25 MW/m^2 .² The divertor module is 0.67 m in the toroidal direction and 0.55 m in the poloidal direction and baffle module is 0.63 in toroidal and 0.50 in poloidal direction. The flow direction is chosen to be poloidal so that the power input to each flow channel is equal.³ The divertor has a total of 48, 8 mm ID cooling channels for each module. The baffle has 24 cooling channels of 10 mm diameter.

III. POWER FLOWS

The FIRE tokamak is planned to be operated in four modes: 1) base line D-T (20s), 2) an advanced physics D-D (215 s), 3) a long burn D-T mode (31 s), and 4) a high field operation D-T (12 T, 8 MA, 12 s). The heat loads on the outer divertor and baffle are highest during the base line D-T operation with power flows as shown in Table I.

Table I. The power flows during baseline D-T mode

	Outer Divertor	Baffle
Total Power (MW)	34.3	10.7
Peak Power/module (MW)	2.32	0.58
Peak Heat Flux (MW/m^2)	20.0	6.00
Nuclear heating in Tungsten (W/cm^3)	42	34
Nuclear heating in Cu (W/cm^3)	16	13

IV. THERMAL HYDRAULIC ANALYSIS

The thermal design criteria for divertor and baffle are:

- Water Inlet Temperature = 30°C
- Heat Loads = as shown in Table I.
- Maximum Tungsten Temperature = 1800°C
- Maximum Copper Temperature = 500°C
- Minimize inlet pressure, flow rate and pumping power
- Safety Margin on CHF = 1.5

In order to remove an incident heat flux of 20 MW/m^2 , a very large flow velocity ($> 20 \text{ m/s}$) is required if smooth channels are used. The flow velocity and flow rate required to cool the outer divertor can be reduced by using a heat transfer enhancement in the flow channels. A review of enhancement methods³ shows that a swirl tube (ST) is the best available method. The ST is easy to fabricate and has a large reliable database. For a

ST with a tape thickness of 1.5 mm and a twist ratio of 2 in the divertor channels of 8 mm diameter, a flow velocity of 10 m/s gives sufficient safety margin on CHF for the divertor. If two adjacent channels are connected in series, the maximum outlet temperature is 95°C and minimum exit pressure is 1.05 MPa, resulting in a minimum subcooling of 87°C .

A two dimensional FE analysis of a divertor cell was performed for these flow conditions. The divertor cell consists of a copper mono block $14 \text{ mm} \times 15 \text{ mm}$ with the 5 mm tungsten brush as PFC. One mm long conical tip of the tungsten rods is embedded in the copper heat sink by high isostatic pressure (HIP) process. An effective thermal conductance of the tungsten copper interphase was determined by a 3-D finite element analysis with COSMOS code.⁴ Figure 2 shows the result. The heat transfer coefficient in the coolant channel is calculated as a function of wall temperature over forced convection, nucleate boiling and post CHF region by the method described in Ref. 3. The pressure drop is calculated by Lopina-Bergles correlation.⁵

Figure 3 shows the temperature distribution at the end of 20 s for the divertor subjected to an incident heat flux of 20 MW/m^2 and nuclear heating as shown in Table I. The peak surface temperature is 1585°C and the maximum copper temperature is 488°C . Based on the flow velocity of 10 m/s, the flow per module is 9 ℓ/s . Figure 4 shows the transient of the peak surface temperature. A steady state is reached in 6 s. The peak temperatures will be lower by 25 to 50°C when axial distribution of heat flux is available and is modeled in the 3D thermal analysis. Similar analysis was performed for the baffle. Due to lower peak heat flux of 6 MW/m^2 , smooth channels can be used in this region. Use of heat transfer enhancement is not justified in the baffle region. The

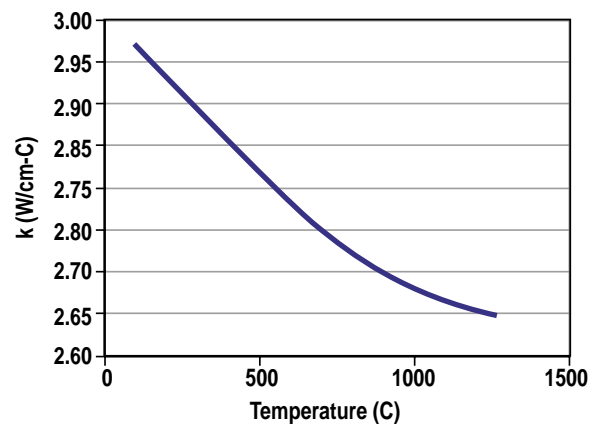


Fig. 2. Effective thermal conductivity of tungsten copper interphase.

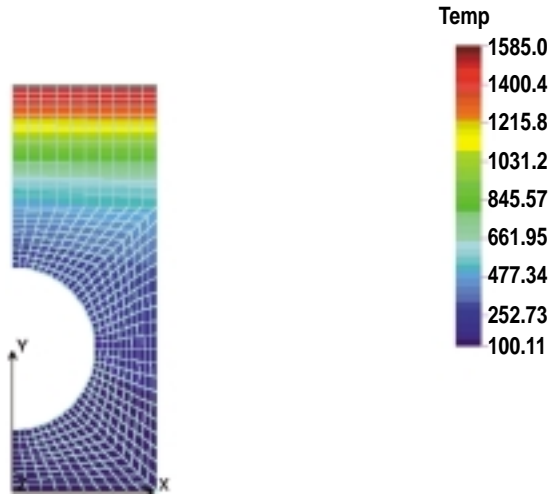


Fig. 3. Temperature (°C) distribution in divertor at the end of 20 s pulse with 20 MW/m² heat flux and heat generation shown in Table I.

additional fabrication cost is not justified by very small savings in total flow rate and pumping power. The required flow velocity in baffle channels is 3 m/s. For two channels connected in series, flow per module is 3.5 ℓ/s, the coolant temperature rise is 43.3°C and the pressure drop is 16.8 kPa. The results of the thermal analyses are summarized in Table II.

V. CRITICAL HEAT FLUX

The FE analysis described above indicated that the peak wall heat flux (heat flux on coolant channel wall) in the divertor is 30.55 MW/m². The wall critical heat flux under these flow conditions, calculated by Baxi correlation³ is 45 MW/m². Hence the safety margin on CHF in the divertor region is about 1.5. Similar calculations in the baffle region show a safety margin of 1.9.

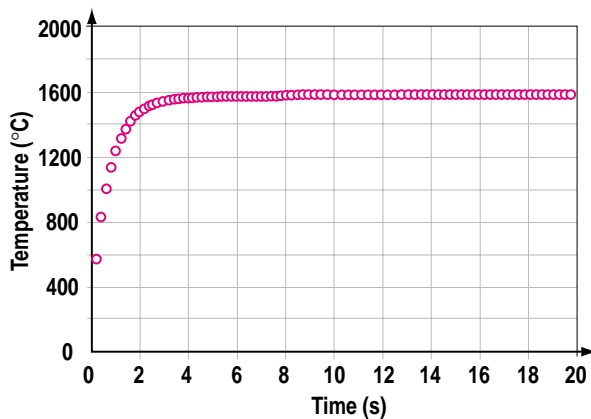


Fig. 4. Variation of Peak Temperature (°C) in Divertor.

Table II. Results of Thermal Hydraulic Analysis

	Outer Divertor	Baffle
Peak Heat Flux(MW/m ²)	20	6
Channel Diameter (mm)	8	10
Pitch (mm)	14	21
Number per Module	48	30
Number in Series	2	2
Enhancement	ST, t=1.5 mm Y= 2	no
Maximum PFC Temperature (°C)	1585	738
Maximum Copper Temperature (°C)	488	404
Flow Velocity (m/s)	10	3
Flow/Module (ℓ/s)	9	3.5
Exit coolant Temperature (°C)	95	73.3
Exit Pressure(MPa)	1.06	1.48
Exit Subcooling (°C)	87	120
Wall CHF (MW/m ²)	45.	12.1
Maximum Wall Heat Flux (MW/m ²)	30.6	6.31

VI. CONCLUSIONS

A satisfactory thermal hydraulic design of the FIRE divertor and baffle can be achieved with existing technology. At an inlet of 30°C, 1.5 MPa and a flow velocity of 10 m/s in a swirl tube, water cooling provides a safety margin of 1.5 on the CHF to remove 20 MW/m² of peak heat flux on the divertor surface. The total flow rate required for the outer divertor is 288 ℓ/s and for the baffle is 112 ℓ/s. The peak PFC temperature is 1585°C and peak copper temperature is 488°C.

ACKNOWLEDGMENT

Work supported by U.S. Department of Energy under Contracts DE-AC03-98ER54411, DE-AC02-76-CH03073, and DE-AC04-94AL85000.

REFERENCES

1. R.J. Thome, “Engineering Overview of Fusion Ignition Research Experiment (FIRE),” Symp. on Fusion Energy, Albuquerque, New Mexico, 163 (1999).
2. D.E. Driemeyer, “Development of Direct HIP-Bonding Processes for Tungsten-Brush Armor Joining,” Symp. on Fusion Energy, Albuquerque, New Mexico, 369 (1999).
3. C.B. Baxi, “Thermal Hydraulics of Water Cooled Divertors,” Presented at the SOFT, Madrid, Spain, 2000.
4. COSMOS, A Finite Element Analysis Code, Structural Research, Santa Monica, California.
5. R.F. Lopina and A.E. Bergles, “Heat Transfer in and Pressure Drop in Tape Generated Swirl Flow of Water,” J. Heat Transfer, Vol. 91, 434 (1969).